

85028

18.7100 1.2400

S/094/60/000/010/002/002

E073/E335


AUTHORS: Strakhov, K.I., Kachanov, I.I. and Yakovlev, V.A.

TITLE: High-temperature Induction Furnace of Industrial  
Frequency for Brazing of Components

PERIODICAL: Promyshlennaya energetika, 1960, No. 10,  
pp. 15 - 16

TEXT: The authors propose a new design of an induction type furnace operating at the supply frequency with a permanent hermetically closed muffle and not an expendable one. The furnace forms a coreless transformer, the primary winding of which is a multiturn solenoid (inductor) and the secondary winding is the hermetic muffle made of a refractory metal. On connecting the inductor to an AC supply, currents are induced in the muffle and partly also in the components inside it which generate the required brazing heat. The temperature control is effected by means of a potentiometer on the basis of temperature values derived from thermocouples fitted inside the furnace. The furnace consists of the following basic parts: housing - 1; inductor - 2; thermal insulation - 3; hermetic muffle - 4; lid - 5 and the magnetic circuit - 6.

Card 1/3



S/094/60/<sup>85028</sup>000/010/002/002  
E073/E335

High-temperature Induction Furnace of Industrial Frequency for  
Brazing of Components

The housing is made of ordinary "steel 5"<sup>16</sup> and its dimensions are 1 000 x 1 000 mm. To prevent heating of the housing separation gaps are provided. The inductor is a two-layer one and has 78 turns of a 16 x 16 mm hollow aluminium conductor. The outer layer has 5 tappings, enabling selection of the necessary thermal regime of the furnace. The dimensions of the inductor are: external diameter 823 mm; internal diameter 785 mm and height 750 mm. The thermal insulation is made of "ultra-lightweight" material (between the internal layer and the external surface of the muffle) and firebrick. The muffle is made of refractory  $\text{ЭИ-435}$  (EI-435)<sup>18</sup> sheet steel, 11 mm thick; the joints are fused by argon arc welding. The cover of the furnace is of nonmagnetic steel, 14 mm thick with a pipe connection for fitting a vacuum pump, introducing a gas flux and thermocouples. On the inside the lid is fitted with thermal insulation. On the outside it is water-cooled. The furnace characteristics are as follows: power 65 kW; voltage 380 V; current consumption 180 A; current intensity  
Card 2/3


85028

S/094/60/000/010/002/002

E073/E335

High-temperature Induction Furnace of Industrial Frequency for  
Brazing of Components

in the furnace 700 A; rating of the condenser bank 350 kVAR;  
temperature 1 200 - 1 250 °C. This furnace has the following  
advantages: the power consumption is only one-quarter of  
that of a chamber furnace; the process is much less laborious;  
a great saving is obtained in expensive refractory metal for  
manufacturing the muffles. The annual saving in electricity  
amounts to 600 000 kWh. This proposal was awarded second  
prize in the Fifteenth All-Union Competition for Saving  
Energy. There is 1 figure.



Card 3/3

TRET'YAKOV, Yu.D.; KACHANOV, I.N.

Isothermal solubility diagram for the quaternary system  $\text{FeSO}_4 \cdot (\text{NH}_4)_2\text{SO}_4$ -  
 $\text{MgSO}_4 \cdot \text{MnSO}_4 \cdot (\text{NH}_4)_2\text{SO}_4$ - $\text{ZnSO}_4 \cdot (\text{NH}_4)_2\text{SO}_4$ - $\text{H}_2\text{O}$  at 40°C. Zhur. neorg. khim.  
7 no:7:1708-1715 J1 1962. (MIRA 16:3)  
(Systems (Chemistry)) (Solubility) (Sulfates)

KACHANOV, K.S., inzh.

Work practices of the organizations of the Main Administration  
for the Erection of Metallurgical Plants. Mont. i spets. rab.  
v stroi 25 no.11:3-8 N '63. (MIRA 17:1)

1. Glavnoye upravleniye po montazhu metallurgicheskikh predpriyatiy.

GLIKMAN, L.A., prof., doktor tekhn.nauk; KACHANOV, L.M., prof., doktor fiz.-  
matem.nauk

"Residual Stresses" by I.A.Birger. Zav.lab. 30 no.12:1523-1524 '64.  
(MIRA 18:1)

*KACHANOV, L. M.*

KACHANOV, L. M.

Variatsionnye printsipy dlia uprugo-plasticheskikh sredin. (Prikladnaia matematika i mekhanika, 1942, v. 6, no. 2/3, p. 187-196, bibliography)

Title tr.: Variational principles for elastic-plastic solids.

QA801. P 7 1942

SO: Aeronautical Sciences and Aviation in the Soviet Union, Library of Congress, 1955.







KACHANOV, L. M.

Aug 1947  
 Creep Stress Analysis  
 "Creep in Combined Stress," L. M. Kachanov, Candidate of Physical-Mathematical Sciences, Central Scientific Research Boiler Turbine Institute 1947, I. I. Pol. 577  
 "Mikroostreymiy" No 1  
 Mathematical discussion illustrated with formulas and a few diagrams on general questions on the theory of creep in combined stress, by means of which the change of strain and stress with time and the elasticity of strain are studied. The principle of least work, a generalized principle of Castiglione, are formulated. These formulas give in a simple form answers to the problems of creep in spherical bodies and cylindrical pipes with thin walls.  
 Aug 1947

PA 24T31

KACHANOV, L.M.; LUR'YE, A.I., prof., red.; LOYTSYANSKIY, L.G., prof., red.;  
DZHANELIDZE, G.Yu., red.; VOLCHOK, K.M., tekhn. red.

[Mechanics of plastic media] Mekhanika plasticheskikh sred. Leningrad,  
Gos. izd-vo tekhniko-teoret. lit-ry, 1948. 215 p. (MIRA II:7)  
(Deformations (Mechanics)) (Elastic solids)

*A. N. KACHANOV, A. M.*

\*Kachanov, L. M. On the mechanics of ...  
... principles and ...

... from J. Appl. Math. Mech. [Akad. Nauk  
SSSR. Zhurnal Prikl. Mat. Mech.] 4, no. 3, 37-42 (1940)  
The author's initials are misprinted L. N. in the title.



*KACHANOV, L. M.*

KACHANOV, L. M.

Nekotorye voprosy teorii polzuchesti. Leningrad, Gostekhizdat, 1949.  
164 p., diags. (Sovremennye problemy mekhaniki).

Bibliography: p. 163-164.

Title tr.: Some problems in the theory of creep of metals.

Reviewed by IU. N. Rabotnov in Sovetskaia kniga, 1951, no. 1, p. 55-57.

TA 460.K23

SO: Aeronautical Sciences and Aviation in the Soviet Union, Library of  
Congress, 1955.

ARCHIVNOV. L.M.

materials of this type is established, and various examples are discussed including flexure and torsion of bars and the general relaxation problem where the displacements are prescribed on part of the surface of a body.

ii Page 100

KACHANOV, L. M.

33895. O Prilozhenii Tyeorii Kirkhgofa-Klyebsha K Voprosam Ustoychivosti Dyeformatsii I Kolyevaniy Odnogo Klassa Plastin I Obolochyek. Uchyen. Zpiski, (Lyeningr. Gos. Un-t Im. Zhdanova), Syeria Matyem. Nauk, VYP 17, 1949, G. 95-102. - Bibliogr: 6 Nazv.

SO: Letopis' Zhurnal'nykh Statey, Vol. 46, Moskva, 1949.



KACHANOV, I. M.

Applying the Kirchhoff-Kiebach theory to problems on the stability,  
deformation and vibration of a specific class of plates and shells.  
Uch.zap.Len.un. no.114:95-102 '49. (MLRA 10:3)  
(Elastic plates and shells)

*Strain*

... torsion,  
Paper presents the plastic torsion of a uniform pris-  
matic bar. The analysis is based upon a total strain theory  
of plasticity in which  $G\gamma = \tau^{1+\beta}$  ( $\gamma$ =engineering shear  
strain,  $\tau$ =shear stress,  $G$  and  $\beta$ =positive constants  
typifying mechanical behaviour of material under shear).  
The objective is principally to discuss cases in which the  
cross-sectional shape exhibits slight deviations from a  
circle. In particular, some results are obtained for a case  
when there is a local notch of special shape.

*H. G. Hopkins (Sevenoaks)*



HT

KACHANOV, L. M.

"Investigations in the Theory of Plasticity." Sub 26 Apr 51, Inst  
of Mechanics, Acad Sci USSR.

Dissertations presented for science and engineering degrees in Moscow  
during 1951.

SO: Sum. No. 480, 9 May 55.

BTR KACHANOV, L.M.

1788: Stability of Plane Forms of Bending Beyond the Elastic Limit. L. M. Kachanov. *Prikladnaya Matematika i Mekhanika*, v. 15, Mar.-Apr. 1951, p. 195-206.  
The theory of small elastic-plastic deformations is used to calculate the above. Results are tabulated and charted.

KACHANOV, L. M.

USSR/Mathematics - Elasticity

Sep/Oct 51

"Stability of Plane Form of Deflection beyond  
Elasticity Modulus (Effect of Strengthening)," L.  
M. Kachanov, Leningrad State U

"Prikl Matem i Mekh" Vol XV, No 5, pp 639, 640

Problem was discussed by author previously (ibid.  
Vol XV, No 2, 1951). Here he considers the case  
in which the material is strengthened linearly.  
In boundary case he obtains stability soln for (1)  
elastic bands, and (2) elastic bands of ideal  
plastic material. Submitted 20 Jan 51.

193760

*17/11/1951*

Kačanov, L. M. Stability of the plane form of bending beyond the elastic limit. III. Influence of complex loading. Akad. Nauk SSSR, Prikl. Mat. Meh. 15, 762-764 (1951). (Russian)

*0300*

It is pointed out that the theory of small elastic-plastic deformations is not strictly applicable to this problem, since the loading is "complex" because the stress components do not increase proportionately to a given parameter. The theory of plastic flow is applied and the solution is shown to be essentially in agreement with previous solutions. The author obtained by employing the former theory (in the three preceding reviews).

*H. I. Ansof.*

Source: Mathematical Reviews,

Vol 13 No. 9

*Handwritten marks*

AMR

26

3838. Kachany, L. M., Stability of a plane deflection form beyond the yield point (in Russian), *Doklady Akad. Nauk SSSR (N.S.)* 77, 6, 986-987, Apr. 1961.

Author gives relations between stress couples and components of curvature and twist for a bent rod of rectangular cross section, valid beyond the yield point (analogous to Kirchhoff's relations in elastic range); these, with the known Kirchhoff-Clebsch's equations (equations of equilibrium and relations between components of curvature and displacements), form a complete system of 15 equations with 15 unknowns. The case of a resistless rod under the von Mises condition of plasticity is discussed, and indications for a more general theory (curved rod, plasticity with work hardening) are given. For solution of the equations, Hils's method is proposed. No numerical results are given, nevertheless, author concludes that S. Timoshenko's proposal to calculate with von Kármán's modulus leads to unsatisfactory results.

J. M. Klitchieff, Yugoslavia

Oct. '91

ASR 314 METALLURGICAL LITERATURE CLASSIFICATION

KACHANOV, L. M.

Strains and Stresses

Stability of arches beyond the limit of elasticity. Vest. Len. un 7 No. 12, 1952.

An analysis of this stability on the basis of plasticity theory in which Young's modulus is replaced by Karman's modulus, as was recommended in the case of plastically-strained arches by A. N. Dinnik, 'Stability of Arches,' (Ustoychivost' Ark.,) State Tech. Press, 1946. States that this is analogous to the case of longitudinal bending of straight compressed beams.

253T110

9. Monthly List of Russian Accessions, Library of Congress, June 1953, Unclassified



KACHANOV, L. M.

PA 249T47

USSR/Physics - Elasticity

1 Feb 53

"Stability of Elastic-Plastic Equilibrium of a Compressed and Twisted Wall," L. M. Kachanov

DAN SSSR, Vol 88, No 4, pp 627-630

Studies stability of a circular wall compressed and twisted beyond the limit of elasticity. The problem as applied to an ideally elastic wall was discussed in previous articles (see Greenhill, Proc. Inst. Mech. Eng. London (1883); E. L. Nikolai, *ibid.* 6,30 (1926) and others). Presented by Acad V. I. Smirnov. Received 20 Oct 52.

249T47

*KACHANOV L.M.*

USSR/Engineering - Steel pipes

Card 1/1 Pub. 128 - 5/32

Authors : Leleev, N. S.; Troyanskiy, E. A.; Zalkind, E. M.; Kats, Sh. N.; Zakharov, A. A.; and Kachanov, L. M.

Title : Comments and critical review of the article, "A Problem Concerning the Strength of Steel Pipes for High-Pressure Boilers"

Periodical : Vest. mash. 11, 24-27, Nov 1954

Abstract : A discussion and rebuttal of the article, "A Problem Concerning the Strength of Steel Pipes for High-Pressure Boilers", written by N. S. Leleev, and E. A. Troyanskiy, is presented. Graphs; table; diagram.

Institution : ...

Submitted : ...

KACHANOV, L. M.

USSR/Theory of elasticity

Card 1/1

Author : Kachanov, L. M.

Title : Problem of deformation of a plastic layer

Periodical : Dokl. AN SSSR, 96, Ed. 2, 249 - 252, May 1954

Abstract : Report is devoted to an axially symmetrical problem of deformation of a plastic layer. Actually there are two problems involved; 1) problem of the stressed state in a plastic seam and the development of a method determining the resistance of pliable metals to breaking away by means of elongation tests of a thin layer of the tested metal fused with solid parts. Consideration must be given to the composite boundary conditions causing displacement on the surface of the contact. Actual test procedure is described. Five references; 1 USSR 1950.

Institution : .....

Submitted : Presented by Academician V. I. Smirnov, March 15, 1954  
February 23, 1954

USSR/Mechanics - Elasticity and Plasticity

FD-2489

Card 1/1      Pub 85-16/19

Author      : Kachanov, L. M.

Title        : On complex loading

Periodical : Prikl. Mat. i Mekh., 19, 371-375, May-June 1955

Abstract    : The author compares the two types of fundamental dependencies used in the theory of plasticity. They are 1) equations of the theory of elastic-plastic deformations which establish a connection between components of deformation and components of stress and 2) equations of the theory of plastic flow, which connect infinitely small increments of these components. The author presents some qualitative indications that results obtained from the two different theories will coincide closely in most cases.

Institution: --

Submitted   : November 24, 1954

~~KACHANOV, Lazar' Markovich; FEL'DMAN, G.I.,~~ redaktor; MURASHOVA, N.Ya.,  
tekhnicheskij redaktor

[Principles of the theory of plasticity] Osnovy teorii plastichnosti.  
Moskva, Gos. izd-vo tekhniko-teoret. lit-ry, 1956. 324 p. (MIRA 10:2)  
(Plasticity)

SECRET

SECRET

KADIA N. B. V. J. M.

status of criteria of stability arriving at the conclusion

status of criteria of stability arriving at the conclusion

hypotheses including Engesser's hypothesis and shows that in the elastoplastic range only Shanley's





KACHANOV, L. M.

Physicists L. M. Kachanov, Ye. I. Edel'shteyn, G. V. Vinogradov, G. N. Kuznetsov, M. P. Volarovich, and A. V. Stepanov and geologists F. I. Vol'fson, V. A. Aprodov, N. I. Borodsyevskiy, and Yu. S. Shikhin -- on the problems of modeling tectonic phenomena.

paper presented at the First All-Union Conference on Tectonophysics, Moscow, 29 Jan - 5 Feb 1957.

Kachanov, L. M.

**AUTHOR:** Ivlev, D. D.

**TITLE:** Conference on Sustained Static Strength of Turbines Working at High Temperatures (Sobremennyye po dlyitel'noy staticheskoy vydnosimosti turbovuzlakh, rabotayushchikh pri vysokoy temperature)

**PERIODICAL:** Izvestiya Akademii Nauk SSSR, Otdel'nyye Tekhnicheskiye Serii, 1958, No 4, pp 149 - 150 (USSR)

**ABSTRACT:** The Commission on the Strength of Gas Turbines from the Institut Matematiki AN SSSR (Institute of Mechanics of the Acad. USSR) (Chairman - Yu. N. Rabotnov) and the Strength Section of the Leningrad Technical Committee on Turbine Construction (Chairman - V. I. Mamonov) held a conference during November 20-22, 1957 on the sustained static strength of turbine components working at high temperatures.

SOV/2a-58-a-31/39  
 Conference on Sustained Static Strength of Turbine Components Working at High Temperatures

G. A. Zhilyakov (TsNITMASH) described the results of an experimental investigation of creep in the boiler steel 121 B1 (T121B1) under complex stress conditions.

L. M. Kachanov (TsNII, L. M. Polzunov) gave a paper on Investigation of Deformation and Sustained Strength of Tubes containing dislocations on the study of creep under complex stress conditions.

A. V. Gribin (Vysshaya Voenno-sukhoye uchebnoye im. Zhukovskogo Aviaatsionnoyeh Inzhenerovskoye Uchebnoye Zavedeniye) read a paper on "Calculation of the 'fir-tree' Roots of Blades of Gas Turbines in the Creep Deformation Regime".

L. M. Kachanov (Leningradskiy gosudarstvennyy universitet - Leningrad State University and TsNII im. Polzunov) dealt with creep under initial plastic deformation, with a view to calculating the deformation state of components made from special heat-resistant steels.

I. M. Kabanov (Moscow State University, Institut Mekhaniki AN SSSR, Institute of Mechanics of the Acad. USSR) described the results of theoretical and experimental investigations in the field of creep under complex stress conditions. He stated that there now exists a theory allowing calculation of the stress and deformation state in turbine disks and rotors at high temperatures. In addition he has designed constructed apparatus for investigating sustained strength and creep of heat resisting alloys under complex stress conditions and a number of valuable results have been obtained with this apparatus.

A. P. Bogdanov (TsNII im. Polzunov) discussed the choice of the nature of loading of components working at high temperatures.

(S. I. Ivanov) gave a paper "On Construction of Factors of Sustained Static Strength" which described results obtained on low-power turbine equipment.

The paper of N. K. Kalinovsky dealt with the bearing capacity of turbine rotors.

Many participants remarked on the increasing need for extensive co-ordination of work in the field of strength of gas turbines.

Card 5/7

122

**AUTHOR:** Kachanov, L.M. and Nemchinskiy, A.L.

**TITLE:** On a method of determining the fracture strength.  
(Ob odnom sposobe opredeleniya soprotivleniya otryvu)

**PERIODICAL:** "Fizika Metallov i Metallovedenie" (Physics of Metals and Metallurgy), 1957, Vol. IV, No. 1 (10), pp. 151-160 (U.S.S.R.)

**ABSTRACT:**

Existing methods of testing the fracture strength of ductile metals, particularly of low carbon steel, have a number of disadvantages. In earlier work (Zav. lab., 1952, Vol. XVIII, 1381), one of the authors described results of fracture tests carried out with cylindrical specimens of high strength steel containing a thin transverse layer of a steel to be investigated; the strength figures were obtained on the assumption that the load distribution was uniform. It was later found that this was not the case, and in this paper formulae are derived which enable one to calculate the real ratio of these stresses. It was found dilatometrically that carbon steel had a linear contraction of 0.228% on cooling down from +20 to -190 °C, whilst hardened chromium-nickel steel containing 0.3% C contracted under the same conditions by 0.236%. This slight difference of 3.5% in the thermal expansion of the two steels affect appreciably the results of the investigations. The authors carried out experiments with specimens which were manufactured by forge welding of a packet consisting of two sheets of chromium-nickel steel with an

AUTHOR: Kachanov, L. M. (Leningrad).

24-5-4/25

TITLE: On plastic bending of curved thin walled tubes.  
(O plasticheskoy izgibe krivyykh tonkostennykh trub).

PERIODICAL: "Izvestiya Akademii Nauk, Otdeleniye Tekhnicheskikh Nauk"  
(Bulletin of the Ac.Sc., Technical Sciences Section,  
1957, No.5, pp. 42-47 (U.S.S.R.))

ABSTRACT: Bending of curved thin walled tubes is accompanied by flattening of their profile and as a result of that, their flexibility increases sharply. A theoretical analysis of this phenomenon was first carried out by Karman in 1911 and a number of authors have dealt with the problem in more recent years. In earlier work of the author (6) attempts were made to analyse bending of curved thin walled tubes in the case of plastic deformations whereby the solution is constructed on the basis of the principle of minimum total energy; the difficulties caused by the non-linearity of the problem were overcome on the basis of the assumption that the relation between the intensities of stresses and strains  $T$  and  $\Gamma$  can be approximated by the parabola  $T = 2A_1 \Gamma - 4A_2 \Gamma^2$ , whereby  $A_1 > 0$ ,  $A_2 > 0$  are constants. Thus,  $dT/d\Gamma > 0$ , so that this

Card 1/3

On plastic bending of curved thin walled tubes. (Cont.)  
approximation allows relatively small deviations from <sup>24-5-4/25</sup> the linear law. It was found that during plastic deformations the flexibility of the tube increases only slightly. In this paper the solution is based on a different variational principle, the principle of minimum additional work; it is in this case possible to arrive at a solution of the problem for cases when  $\Gamma = BT^m$ , whereby B is a constant and m is an odd number larger than zero; flexibility values for other magnitudes of m can be obtained by interpolation. This approximation enables establishment of the fact that the flexibility of the tube increases sharply with increasing m. The given results indicate that the flexibility coefficient  $\delta$  increases sharply on transition into the region of plastic deformations. This is attributed to the fact that flattening causes reduction of the longitudinal deformation and a reduction in the stress  $\sigma$ . For balancing a given bending moment, the required increase in  $\sigma$  is larger in the plastic range of deformations than in the elastic one. The obtained solution can be directly applied to the case of bending of a curved tube under conditions of steady state creep. The limit carrying capacity of such tubes is also considered.

Card 2/3

On plastic bending of curved thin walled tubes. (Cont.)  
There are four figures, 7 references, 4 of which are <sup>24-5-4/25</sup> Slavic.

SUBMITTED: March 1, 1957.

AVAILABLE:

Card 3/3

*K. M. KACHANOV, L. M. KACHANOV*  
AUTHORS: Kats, Sh. N. and Kachanov, L. M. (Leningrad) 24-11-22/31

TITLE: On plastic deformation in the case of complicated loading.  
(O plasticheskoy deformatsii pri sloznom nagruzhenii)

PERIODICAL: Izvestiya Akademii Nauk SSSR, Otdeleniye Tekhnicheskikh Nauk, 1957, No.11, pp. 172-173 (USSR)

ABSTRACT: The results of various authors, for instance, of Neal, B. (Ref.5) relating to the determination of the torsion resistance of an initially bent rod prove indirectly the usefulness of the plastic flow theory. Therefore, the authors considered it of interest to accumulate various experimental data on this problem and here results are described of torsion experiments on tubes which were first plastically deformed by internal pressure. Inside a special set-up a vertically disposed tube was fixed which was stressed by internal hydraulic pressure. The measurement of the pressure was accurate to a degree 0.35%. The change in the tube diameter under the effect of internal pressure was recorded in six points along the circumference of the tube with an indicator having scale divisions of 1  $\mu$ . The torsion was effected by loads applied to arms of 1 m length. Seven Card 1/2 tubes were investigated, all of which were produced from

24-11-22/31

On plastic deformation in the case of complicated loading.

Steel 20 which was first annealed to obtain given mechanical properties. The experimentally determined curves show that in presence of a plastic deformation in the tube, caused by internal pressure, the initial shear modulus will equal the elastic shear modulus as follows from the theory of plastic flow; thereby, the degree of plastic deformation caused by the internal pressure does not manifest itself greatly on the values of the shear modulus and torsion. The coefficient of proportionality between the torque and the twist angle is strongly dependent on the magnitude of accumulated deformation; these conclusions of the theory of elastic-plastic deformations contradict the above mentioned experimental data.

There are one figure and 5 references, one of which is Slavic.

SUBMITTED: May 22, 1957.

ASSOCIATION: Central Boiler-Turbine Institute. (Tsentral'nyy Kotloturbinnyy Institut).

AVAILABLE: Library of Congress.

Card 2/2



KACHANOV, L.M. (Leningrad)

Elastic and plastic equilibrium of a wedge under conditions of flat tension. Prikl.mat. i mekh. 21 no.3:413-418 My-Je '57.

(MIRA 10:10)

(Wedges) (Elasticity)

AUTHOR: Kachanov, L. M. (Leningrad) SOV/24-58-2-5/37  
TITLE: On the Rupture Time in Creep Conditions (O vremeni razrusheniya v usloviyakh polzuchesti)  
PERIODICAL: Izvestiya Akademii Nauk SSSR, Otdeleniye Tekhnicheskikh Nauk, 1958, Nr 8, pp 26-31 (USSR)  
ABSTRACT: Recently Hoff (Ref.1) defined the rupture time of a bar in tension as the time at which the cross-sectional area of the bar vanished as a result of unbounded quasi-viscous flow. Kats (Ref.2) has studied by the same method the strength of thick-walled pipes under an internal pressure. The rupture time of a rotating disc with a hole has been calculated by Rosenbloom (Ref.3). In this paper the author describes an attempt at defining the rupture time taking account of embrittlement. The effect of stress concentration has been evaluated by means of the analysis described. If at a given temperature the material can undergo considerable deformation, the stress distribution is noticeably evened out and hence the effect of a stress concentration on the strength of the material is weakened. Conversely, for small deformations  
Card 1/2 the sensitivity of the material to stress concentration

On the Rupture Time in Creep Conditions

SOV/24-58-8-5/37

is increased. It is also possible to make a comparatively simple approach to the definition of the rupture time and the conclusions generally agree with the observations. In some cases (for instance, inside an aggressive medium) a preferential formation of cracks emanating from the surface of the body was observed; for such cases the time to failure can be determined according to the method described in the paper provided it is supplemented by an equation which determines the speed of failure along the normal to the surface. The author expresses his thanks to I. G. Sobolev and Ya. B. Fridman for their useful advice.

There are 2 figures and 11 references, 8 of which are Soviet, 2 English and 1 German.

SUBMITTED: April 5, 1958

1. Metals--Mechanical properties
2. Metals--Failure
3. Metals--Creep
4. Metals--Stresses

Card 2/2

AUTHOR: Kachanov, L. M. (Leningrad) SOV/179-59-3-13/45

TITLE: Approximate Solution of Steady Creep Problems  
(Priblizhennoye resheniye zadach ustanovivsheysya polzuchesti)

PERIODICAL: Izvestiya Akademii nauk SSSR, Otdeleniye tekhnicheskikh nauk, Mekhanika i mashinostroyeniye, 1959, Nr 3, pp 84-95 (USSR)

ABSTRACT: The paper is a continuation of earlier work (Refs 1 and 5). It is assumed that in steady creep the shear strain and shear stress are connected by the power relationship

$$H = BT^m.$$

The strain components are related to the stress components by the equations (Ref 1)

$$\xi_x = \frac{1}{2} BT^{m-1} (\sigma_x - \sigma) = \frac{\partial \Delta}{\partial \sigma_x}, \dots$$

(1.2)

Card 1/2

$$\eta_{xy} = BT^{m-1} \tau_{xy} = \frac{\partial \Delta}{\partial \tau_{xy}}, \dots$$

Approximate Solution of Steady Creep Problems SOV/179-59-3-13/45

where

$$\Lambda = \frac{B}{m+1} T^{m+1} \quad (1.3)$$

and  $\sigma = 1/3 (\sigma_x + \sigma_y + \sigma_z)$ .

An approximate method of solving creep problems based on these equations is proposed and applied to a hollow sphere under the action of internal pressure; the pure bending of a bar of rectangular cross-section; the torsion of a bar of rectangular cross-section, and the axially symmetric bending of a plate. The results are illustrated graphically (Figs 5-10) and numerically. There are 10 figures and 6 references, 5 of which are Soviet and 1 English.

SUBMITTED: February 27, 1959

Card 2/2

KACHANOV, L.M. (Leningrad)

Variational methods for the solution of plasticity  
problems. Prikl. mat. i mekh. 23 no.3:616-617 My-Je '59.  
(MIRA 12:5)

(Plasticity)

KACHAPOV, L.M.

report presented at the 1st All-Union Congress of Theoretical and Applied Mechanics, Moscow, 27 Aug - 3 Feb '60.

- 130. A. A. Dymov (Moscow): Problems of the theory of plasticity under constant loading.
- 131. V. S. Shalimov (Moscow): Elastic-plastic vibrations of rods of non-linear stress-strain.
- 132. V. S. Shalimov (Moscow): The forced nonlinear flexural vibrations of a homogeneous prismatic rod with a very long rectangular plate.
- 133. G. A. Bilibidze (Moscow): On a method of solving the problem of the stability of a viscoplastic material in the presence of a magnetic field.
- 134. G. A. Bilibidze, V. A. Shalimov (Moscow): An engineering method for the design of open prismatic shells.
- 135. I. I. Zhuravskiy (Leningrad): The distribution of vertical compressive stresses and strains in foundations in homogeneous or stratified soils.
- 136. B. B. Zolotarev (Moscow): Loading of multilayer plates of variable stiffness.
- 137. B. B. Zolotarev (Moscow): The effect of aging and moisture on the strength of concrete.
- 138. L. N. Zhuravskiy (Leningrad): On the use of rupture in creep.
- 139. L. N. Zhuravskiy (Leningrad): On some variational principles in the theory of plasticity.
- 140. L. N. Zhuravskiy (Moscow): A procedure of determining an impact fracture diagram for large deformations.
- 141. G. A. Bilibidze (Moscow): Some generalizations of the formulation of macroscopic and micro-macroscopic plastic problems and methods for their solution.
- 142. A. M. Zhuravskiy (Leningrad): The role of a viscoplastic medium in a shell.
- 143. L. N. Zhuravskiy (Leningrad): On the elastic equilibrium of thin shells by the method of moments.
- 144. L. N. Zhuravskiy (Leningrad): Methods of the influence surface for the stability of the bending stress in thin plates and shells.
- 145. L. N. Zhuravskiy (Leningrad): The stability of a cylindrical shell under a uniform load in a two-dimensional temperature field.
- 146. G. A. Bilibidze (Moscow): Dynamic stability of cylindrical and spherical shells.
- 147. G. A. Bilibidze (Moscow): The influence of initial imperfections on the stability of thin elastic cylindrical shells under uniform axial compression.
- 148. E. S. Zhuravskiy (Moscow): Elastic stability and post-buckling behavior.
- 149. A. A. Kachapov (Moscow): On the stability of shells under the effect of support elasticity on the natural vibrations of rods.
- 150. E. S. Zhuravskiy, L. A. Galka (Moscow): Strength and plasticity of cylindrical shells.
- 151. E. S. Zhuravskiy (Moscow): The design of flange plates and bases on elastic foundations.
- 152. E. S. Zhuravskiy (Moscow): Loading of rectangular shallow shells with elastic ribs.
- 153. E. S. Zhuravskiy (Moscow): On the solution of the nonlinear algebraic equations of shell theory.
- 154. E. S. Zhuravskiy, E. S. Zhuravskiy (Leningrad): On the non-linear interaction of a medium with a shell.
- 155. E. S. Zhuravskiy (Moscow): The elastic equilibrium of a cylindrical plate with a finite number of elliptical holes.
- 156. E. S. Zhuravskiy (Leningrad), E. S. Zhuravskiy (Moscow): The stability of shells in dry friction.
- 157. E. S. Zhuravskiy (Leningrad): Internal stability of elastic arches and shells.
- 158. E. S. Zhuravskiy, E. S. Zhuravskiy (Moscow): Propagation of shells, viscoplastic waves in bars.
- 159. E. S. Zhuravskiy (Moscow): The investigation of critical problems in the theory of elasticity by the method of singular integral equations.
- 160. E. S. Zhuravskiy (Moscow): The formulation of the differential equations of shells on shells by the Rayleigh method.
- 161. E. S. Zhuravskiy (Moscow): The investigation of non-linear viscoplastic problems in some problems of the theory of elastic plates and shells.
- 162. E. S. Zhuravskiy (Moscow): The investigation of non-linear propagation of plastic waves.

SHIKHOBALOV, S.P., otv.red.; GUTMAN, S.G., red.; KACHANOV, L.M., red.;  
KRASNOV, V.M., red.; MAKUTOVA, T.D., red.; PRIGOROVSKIY, N.I.,  
red.; PROSHKO, V.M., red.; ROZANOV, N.S., red.; EDEL'SHTEYN,  
Ye.I., red.; SECHUMELEVA, Ye.V., red.; VODOLAGINA, S.D., tekhn.red.

[Polarization optical method for stress analysis; proceedings of the  
conference of February 13-21, 1958] Poliarizatsionno-opticheskiy  
metod issledovaniya napriazhenii; trudy konferentsii 13-21 fevralia  
1958 goda. Leningrad, Izd-vo Leningr.univ., 1960. 450 p.  
(MIRA 13:6)

(Strains and stresses) (Optical measurements)



PHASE I BOOK EXPLOITATION

SOV/4494

Kachanov, Lazar' Markovich

Teoriya polzuchesti (Theory of Creep) Moscow, Fizmatgiz, 1960. 455 p. 6,500 copies printed.

Ed.: G.I. Fel'dman; Tech. Ye. A. Yermakova.

**PURPOSE:** This book is intended for technical personnel, engineers and designers concerned with metallic creep.

**COVERAGE:** The author gives an account of the basic theory of creep of metals. He examines creep according to the comparatively simple scheme of quasi-viscous flow. Considerable space is allotted to the working out of simple approximate solutions of problems which, according to the author, are of immediate practical interest. Along with the solutions the author gives graphs, examples, and instructions for applying the methods of calculation. Recognition is given to V.I. Rozenblyum, A.N. Grubin, and G.V. Ivanov for help in compiling the book. There are 159 references: 85 Soviet, 68 English, 5 German, and 1 French.

Card 1/1

KACHANOV, L.M. (Leningrad)

Rupture time under creep conditions. *Izv. AN SSSR. Otd. tekhn. nauk. Mekh. i  
mashinostr. no. 5:88-92 S-O '60. (MIRA 13:9)*  
(Creep of materials)

88526

S/179/60/000/006/022/036  
E081/E135

10.9210

AUTHOR: Kachanov, L.M., (Leningrad)  
TITLE: Creep of a Thin Layer Compressed by Rigid Plates  
PERIODICAL: Izvestiya Akademii nauk SSSR, Otdeleniye tekhnicheskikh nauk, Mekhanika i mashinostroyeniye, 1960, No. 6, pp. 138-140

TEXT: The paper is a continuation of previous work (Ref.2). The well known Prandtl solution (see e.g. Ref.1) of the problem of the flow of an ideally plastic layer compressed by rough, rigid plates forms the basis of a number of methods for the approximate calculation of the compression of a plastic layer. The present note derives a solution of the creep problem for a thin layer compressed by rigid rough plates at sufficiently high pressures. Under plane deformation conditions the deformation velocity  $\dot{\epsilon}_z = 0$  and the flow occurs in the  $xy$  plane; the layer occupies the region  $x > 0$ ,  $|y| \leq h$ ; and each of the plates moves into the layer with velocity  $c$ . On the basis of the Coulomb law, the shear stress  $\tau_{xy}$  at the surface of contact is given by:

Card 1/3

88526  
S/179/60/000/006/022/036  
E081/E135

Creep of a Thin Layer Compressed by Rigid Plates (1.1)

$$\tau_{xy}|_{y=\pm h} = \pm k$$

where  $k$  is the yield value, and for slow flow the differential equation of equilibrium is:

$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} = 0, \quad \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} = 0 \quad (1.2)$$

The relation of steady creep is taken in the form:

$$\zeta_x = f(T^2) (\sigma_x - \sigma), \quad \zeta_y = f(T^2) (\sigma_y - \sigma), \quad \eta_{xy} = 2f(T^2) \tau_{xy} \quad (1.3)$$

The deformation velocities  $\zeta_x$ ,  $\zeta_y$ ,  $\eta_{xy}$  are connected with the velocities  $v_x$ ,  $v_y$  by the known formulae:

$$\zeta_x = \partial v_x / \partial x, \quad \zeta_y = \partial v_y / \partial y, \quad \eta_{xy} = (\partial v_x / \partial y) + (\partial v_y / \partial x) \quad (1.4)$$

Card 2/3

88526

S/179/60/000/006/022/036  
E081/E135

## Creep of a Thin Layer Compressed By Rigid Plates

and the equation of incompressibility is  $\xi_x + \xi_y = 0$ . As in the Prandtl solution, it is assumed that  $\tau_{xy} = k\eta$ ,  $\sigma_y = \psi(\xi)$ , where  $\xi = x/h$ ,  $\eta = y/h$ , and  $\psi(\xi)$  is the function to be found. These assumptions, together with Eq.(1.3), lead to:

$$f\left(\frac{1}{4}s^2 + k^2\eta^2\right) = \frac{2c}{s} \quad (2.8)$$

in which  $s = \sigma_x - \sigma_y$ . The roots  $s_*$  of this equation are determined numerically or graphically. We obtain finally:

$$\sigma_y = -k\xi - \int_0^1 s_*(k\eta, c) d\eta; \quad \sigma_x = \sigma_y + s_*(k|\eta|, c)$$

The equations show that with increasing velocity  $c$ , the roots  $s_*$  increase and the contact pressure also increases. If the creep law is linear,  $s_*$  does not depend on  $\eta$ . There are 2 Soviet references.

SUBMITTED: June 6, 1960

Card 3/3

89400

S/040/61/025/001/021/022  
B125/B204

10.9210

16.7300

AUTHOR:

Kachanov, L. M. (Leningrad)

TITLE:

Solving the problem of non-steady creeping

PERIODICAL: Prikladnaya matematika i mekhanika, v. 25, no. 1, 1961, 162-163

TEXT: The present report describes a method of determining more exact solutions (i.e. more exact than the first approximation) of the problems mentioned in the title. P. S. Kuratov and V. I. Rozenblvum (Ref. 1) recently investigated (Ref. 1) a numerical method of solving such problems. Still earlier, the author of the present paper suggested a simple approximation method, which is based upon determining the minimum of the

additional power  $\Lambda + \frac{\partial \Pi}{\partial t} = \min$  of the body in the form

$\sigma_{ij} = \sigma_{ij}' + \tau(t)(\sigma_{ij}'' - \sigma_{ij}') = \sigma_{ij}^{(0)}$  ( $i, j = 1, 2, 3$ ) (2). ( $\Lambda$  - the additional spread of the body,  $\Pi$  - the elastic potential energy of the body,  $t$  - time,  $\sigma_{ij}'$  - the components of the stress,  $\sigma_{ij}'$ ,  $\sigma_{ij}''$  - the components of the stress in the elastic problem and in the problem of steady creeping

Card 1/4

89400

S/040/61/025/001/021/022

B125/B204

Solving the problem of...

respectively). The function  $\tau(t)$  increases monotonously and tends towards unity with  $t \rightarrow \infty$ . The solution (2) is essentially a first approximation. The present article describes, as already mentioned, the determination of more exact solutions. The solution of the problem

$\Lambda + \frac{\partial \Pi}{\partial t}$  is set up in the form  $\delta_{ij} = \delta_{ij}^{(0)} + \sum_{k=1,2,\dots} c_k(t) \delta_{ij}^{(k)}$  (3).  $\delta_{ij}^{(k)}$

here denotes the particular solutions of the homogeneous differential equations of equilibrium, which satisfy the zero-boundary conditions on  $S_F$ ,  $c_k = c_k(t)$ . In this case, any functions of time are concerned. The solution  $\delta_{ij}^{(0)}$  is best separated, because it is a good approximation.

The additional power of the body is now a function of  $\tau$  and  $c_k$ , and the necessary conditions for the minimum lead to a system of differential

equations  $B(t) \frac{\partial \Lambda_1}{\partial c_1} + \frac{\partial^2 \Pi}{\partial \tau \partial c_1} \frac{d\tau}{dt} + \sum_{k=1,2,\dots} \frac{\partial^2 \Pi}{\partial c_1 \partial c_k} \frac{dc_k}{dt} = 0$  ( $t=1,2,\dots$ ) (4).

The curves of creeping are here assumed to be similar, and then  $\Lambda = B(t) \Lambda_1$

Card 2/4

.....

89400

S/O40/61/025/001/021/Q22  
B125/B204

Solving the problem of...

holds, where  $\Lambda_1$  is a function of the stresses alone. The initial conditions for  $c_k$  are  $c_k = 0$  with  $t = 0$ . The second derivations in the equations

(4) are constant, and for the determinant  $\Delta = \left| \frac{\partial^2 \pi}{\partial c_i \partial c_k} \right| > 0$  holds. If the condition just given holds, the system (4) may always be brought to the

normal form  $\frac{dc_i}{dt} = -\frac{1}{\Delta} \sum_{k=1,2,\dots} \Delta_{ik} \left[ B(t) \frac{\partial \Lambda_1}{\partial c_k} + \frac{\partial^2 \pi}{\partial \tau \partial c_k} \frac{d\tau}{dt} \right]$  ( $i=1,2,\dots$ ) (7), where

$\Delta_{ik}$  is the algebraic complement of the corresponding element of the determinant  $\Delta$ . If the solution of the variation equation  $\delta \Delta_1 = 0$  is set up in the form (3), the values  $\tau = 1, c_i = 0, (i = 1,2,\dots)$  correspond to the

exact solution, and here  $\frac{\partial \Lambda_1}{\partial c_i} = 0, \delta^2 \Lambda_1 = \sum_{i,j=1,2,\dots} \frac{\partial^2 \Lambda_1}{\partial c_i \partial c_j} \delta c_i \delta c_j > 0$  (9) holds. ✓

For  $\sigma_{ij} = \sigma_{ij}^0$ , the additional deviation from  $\Lambda_1$ , according to consideration  
Card 3/4



89400

Solving the problem of...

S/040/61/025/001/021/022  
B125/B204

tions made by R. Khill [abstracter's note: probably Hill], attains the absolute minimum, and the solution given here is unique. Finally, the behavior of the functions  $c_k(t)$  in the case of high values of  $t$  are considered. The additional spread of the body is limited, and also the stresses in a sufficiently large volume are limited. Also the functions  $c_k(t)$  may be assumed to be limited. The equation (7) in integral form

reads 
$$-\int_0^{c_i} \left( \sum_{k=1,2,\dots} \Delta_{ik} \left[ B(t) \frac{\partial \Delta_1}{\partial c_k} + \frac{\partial^2 \eta}{\partial c_k \partial t} \frac{d\tau}{dt} \right] \right)^{-1} dc_i = \frac{1}{\Delta} \int_0^t dt \quad (i=1,2,\dots) \quad (10).$$

System (9) is best numerically or graphically solved by means of the Euler method. The method discussed may easily be applied to relaxation problems, mixed problems, to non-uniformly heated bodies, to cases with lacking similarity of creep curves, and to elastic-plastic deformations. There are 4 Soviet-bloc references.

SUBMITTED: October 8, 1960

Card 4/4

KACHANOV, L.M.

Note to L.M.Kachanov's article "Creep of a thin layer compressed by rigid plates." Izv.AN SSSR.Otd.tekh.nauk.Mekh.i mashinostr. no,2: 172 Mr-Ap '61. (MIRA 14:4)

(Creep of materials)

KACHANOV, L.M. (Leningrad)

Kinetics of the growth of cracks. Prikl. mat. i mekh. 25  
no.3:498-502 My-Je '61. (MIRA 14:7)  
(Strength of materials)

KACHANOV, L.M.

Example of the solution of the problem of elastoplastic torsion  
by means of the variational method. Issl.po uprug.i plast.  
no.1:157-161 '61. (MIRA 15:2)

(Torsion)

DIKOVICH, Igor' Leonidovich; KACHANOV, L.M., prof., doktor fiz.-  
mat. nauk, retsenzent; FILIN, A.P., prof., doktor tekhn.  
nauk, retsenzent; NOVOZHILOV, V.V., red.; KUSKOVA, A.I.,  
red.; SHISHKOVA, L.M., tekhn. red.

[Dynamics of elastoplastic beams]Dinamika uprugo-plastiches-  
skikh balok. Leningrad, Sudpromgiz, 1962. 291 p.

(MIRA 15:10)

(Beams and girders)

42101

B.6100

S/179/62/000/005/003/012  
EO31/E135

AUTHOR: Kachanov, L.M. (Leningrad)

TITLE: On the stressed state of a plastic filling

PERIODICAL: Akademiya nauk SSSR. Izvestiya. Otdeleniye  
tekhnicheskikh nauk. Mekhanika i mashinostroyeniye,  
no.5, 1962, 63-67

TEXT: The problem of the tension of an axisymmetric filling was studied in earlier work of the author. In this paper the flow of a plastic filling in conditions of plane deformation under tension (and compression) and the problem of strengthening are considered. The filling is assumed to be thin and joining two sufficiently rigid parts which have approximately the same moduli of elasticity as the filler, but considerably higher yield point values. If a moderate force is applied per unit length perpendicular to the sandwich construction, all the joints will suffer elastic deformation. It is assumed that the surfaces of contact between the filling and the outer layers remain plane. An approximate solution is sought for the stress on the surface of the filling. An expression is derived relating the mean stress  $p$

Card 1/3

On the stressed state of a plastic... S/179/62/000/005/003/012  
E031/E135

to the parameter

$$C = - \frac{1}{2} \frac{v''(\delta)}{v'(\delta)}$$

where  $v$  is the displacement in the axial direction and  $\delta$  is the ratio of the thickness of the filling to its width. For  $C = 0$ , corresponding to the onset of plastic flow,  $p = 2k$  ( $k$  - yield point). For small  $C$  the tangential stress distribution is nearly linear. As  $C$  tends to infinity the tangential stress tends to the constant value  $k$ . This corresponds to the boundary conditions of the layer for which the Prandtl solutions were obtained. For  $C \rightarrow \infty$  the derived formulae yield the well known Prandtl equations. The normal stress component has a maximum value on the axis of the filling. The relation between this maximum value and  $p$  is depicted graphically. To examine the effect of strengthening the material it is assumed that for tangential stresses  $T \leq k$  the material is elastic but that for  $T \geq k$  a linear strengthening applies, i.e. for  $T \leq k$ ,  $T = G\Gamma$  and for  $T \geq k$ ,  $T = A + B\Gamma$ , where  $\Gamma$  is the intensity of the shear deformation and  $G$  is the shear modulus. The resulting

Card 2/3

On the stressed state of a plastic.. S/179/62/000/005/003/012  
E031/E135

changes in the various expressions previously derived are considered. As  $B$  tends to  $G$ ,  $C$  tends to zero. As  $C$  tends to  $C' = (G/B - 1) G$ ,  $p/2k$  tends to infinity. With strengthening there is no limiting load, the quantity  $p/2k$  being bounded only by the strength.

There are 5 figures.

SUBMITTED: June 20, 1962

Card 3/3



*Kachanov, L. M.*

PHASE I BOOK EXPLOITATION SOV/6201

23

Vsesoyuznyy s"yezd po teoreticheskoy i prikladnoy mekhanike. 1st, Moscow, 1960.

Trudy Vsesoyuznogo s"yezda po teoreticheskoy i prikladnoy mekhanike, 27 yanvarya -- 3 fevralya 1960 g. Obzornyye doklady (Transactions of the All-Union Congress on Theoretical and Applied Mechanics, 27 January to 3 February 1960. Summary Reports). Moscow, Izd-vo AN SSSR, 1962. 467 p. 3000 copies printed.

Sponsoring Agency: Akademiya nauk SSSR. Natsional'nyy komitet SSSR po teoreticheskoy i prikladnoy mekhanike.

Editorial Board: L. I. Sedov, Chairman; V. V. Sokolovskiy, Deputy Chairman; G. S. Shapiro, Scientific Secretary; G. Yu. Dzhanelidze, S. V. Kalinin, L. G. Loytsyanskiy, A. I. Lur'ye, G. K. Mikhaylov, G. I. Petrov, and V. V. Rummyantsev; Resp. Ed.: L. I. Sedov; Ed. of Publishing House: A. G. Chakhirev; Tech. Ed.: R. A. Zamarayeva.

Card 1/6

Transactions of the All-Union Congress (Cont.)

SOV/6201

25

**PURPOSE:** This book is intended for scientific and engineering personnel who are interested in recent work in theoretical and applied mechanics.

**COVERAGE:** The articles included in these transactions are arranged by general subject matter under the following heads: general and applied mechanics (5 papers), fluid mechanics (10 papers), and the mechanics of rigid bodies (8 papers). Besides the organizational personnel of the congress, no personalities are mentioned. Six of the papers in the present collection have no references; the remaining 17 contain approximately 1400 references in Russian, Ukrainian, English, German, Czechoslovak, Rumanian, French, Italian, and Dutch.

**TABLE OF CONTENTS:**

**SECTION I. GENERAL AND APPLIED MECHANICS**

• Artobolevskiy, I. I. Basic Problems of Modern Machine Dynamics	5
• Bogolyubov, N. N., and Yu. A. Mitropol'skiy. Analytic Methods of the Theory of Nonlinear Oscillations	25
Card 2/6	

Transactions of the All-Union Congress (Cont.)

SOV/6201

Kachanov, L. M. On Some Variational Principles and Methods in the Theory of Plasticity	358
Kupradze, V. D. The Singular Integral Equation Method in the Spatial Theory of Elasticity	374
Rabotnov, Yu. N. Creep	384
Florin, V. A. Present State and Future Problems in the Mechanics of Soils	396
Sherman, D. I. Two- and Three-Dimensional Problems in the Static Theory of Elasticity	405

AVAILABLE: Library of Congress

SUBJECT: Physics

Card 6/6

IS/dmp/mas  
2-13-62

KACHANOV, L.M. (Leningrad)

Stressed state of a plastic interlayer. Izv. AN SSSR. Otd. tekhn. nauk. Mekh.  
i mashinostr. no. 5:63-67 S-O '62. (MIRA 15:10)

(Deformations (Mechanics))

GRIGORYAN, S.S.; GRIB, A.A.; ZVOLINSKIY, N.V.; KACHANOV, L.M.; PETROSHEN, G.I.

E.I. Shemiakin's article "Expansion of a gas cavity on a noncompressible elastoplastic medium; study of the action of an explosion on the ground" and N.S. Medvedeva and E.I. Shemiakin's article "Strain waves caused by underground explosion in rocks. Izv. AN SSSR. Otd. tekhn. nauk. Mekh. i mashinostr. no. 5:173-177 S-0 '62. (MIRA 15:10)  
(Explosions) (Shemiakin, E.I.) (Medvedeva, N.S.)

Kachanov, L. M. (Prof)

Analytical Methods of Creep Design, Especially Within the Nonlinear Range.

THIRD SYMPOSIUM ON NAVAL STRUCTURAL MECHANICS

INTERNATIONAL CONFERENCE ON ELEVATED TEMPERATURE MECHANICS

NEW YORK, N, Y,

First Day: January 23, 1963

KACHANOV, L.M. (Leningrad)

Shear and compression of a thin plastic layer. Izv. AN SSSR  
Otd. tekhn. nauk. Mekh. i mashinostr. no.2:172-173 Mr-Apr '63.  
(MIRA 16:6)

(Elasticity)

KACHANOV, L.M.

Kinetic theory of the increase of cracks. Issl. po uprav. i plast.  
no.2:66-73 '63. (MIRA 16:8)

(Deformations (Mechanics))



KACHANOV, L.M. (Leningrad)

Creep of a thin layer subjected to compression and bending. Izv.  
AN SSSR. Mskh. i mashinostr. no.4:86-91 J1-Ag '63. (MIRA 17:4)

KACHANOV, L.M. (Leningrad)

"Variational methods in the theory of plasticity"

report presented at the 2nd All-Union Congress on Theoretical  
and Applied Mechanics, Moscow, 29 Jan - 5 Feb 64.

ACCESSION NR: AT4034319

S/2753/64/000/003/0052/0061

AUTHOR: Kachanov, L. M.

TITLE: Bending of an elastic-plastic lamina

SOURCE: Leningrad. Universitet. Matematiko-mekhanicheskiy fakul'tet. Issledovaniya po uprugosti i plastichnosti, no. 3, 1964, 52-61

TOPIC TAGS: lamina, elastic lamina, plastic lamina, lamina bending, bending stress, planar deformation, laminar fastening

ABSTRACT: The author calls attention to the fact that the stress state of thin plastic laminas, used in the fastening of rigid parts, is distinguished by a number of specific features of great practical importance. After noting in passing that the problem of the elongation (compression) of the thin lamina has been studied elsewhere, the author takes up the question of bending in the present article. Specifically, he considers the problem of the bending of a soft plastic lamina under conditions of planar strain (deformation). It is assumed that the lamina is thin; that is,  $\delta = \frac{h}{r} \ll 1$ . The twisting moment  $M$  is referred to unit length in the direction perpendicular to the plane of the drawing. The yield point of the lamina material  $k$  is, moreover, assumed to be

Card 1/3

ACCESSION NR: AT4034319

considerably lower than the yield point of the "rigid" parts, connected by the lamina. In his discussion, the author assumes the elastic properties of both parts to be identical and accepts the condition of incompressibility. The flow in the plastic zones is constricted by the contact surfaces. This is expressed in the origination of tangential stresses on the contact surfaces, since there is no slippage along the latter. Since the lamina is thin, the contact surfaces remain planar, while the section  $\gamma = 0$  remains in plane symmetry; the author therefore accepts the hypothesis of plane sections. The elastic kernel and plastic zones are considered and the corresponding equations are derived. The arbitrary constants  $f_1, c, d$  are determined from the conditions of stress continuity when  $f = f_1$  and the condition of static equivalence of the stresses  $\sigma_y$  to the bending moment  $M$ . Particular cases in which  $q$  is equal to and greater than 1 are considered, and tabular results are given for the numerical solution of the pertinent complex transcendental equation. The author concludes with a brief discussion of the limiting state in the bending of the lamina. "The author takes the opportunity to express his gratitude to O. A. Baksh', whose initiative was responsible for this work, and to A. I. Kuznetsov for a number of valuable comments." Orig. art. has: 6 figures and 36 formulas.

Card 2/3

ACCESSION NR: AT4034323

S/2753/64/000/003/0225/0231

AUTHOR: Kachanov, L. M.

TITLE: Failure time under the influence of a liquid metal medium

SOURCE: Leningrad. Universitet. Matematiko-mekhanicheskiy fakul'tet. Issledovaniya po uprugosti i plastichnosti, no. 3, 1964, 225-231

TOPIC TAGS: metal fatigue, failure time, metal flux, molten metal medium, metallic heat carrier, metal diffusion, creep, corrosion, steel failure, metal injection

ABSTRACT: The author calls attention to the increased use, in recent years, of liquid-metal heat carriers, noting that molten fusible metals may exert a considerable effect on the strength of structural metals. The character of this interaction may vary. The present article deals with a strength analysis under creep conditions for metals which exhibit a sufficient degree of resistance to the effect of liquid-metal media so that the problem of interaction need not be considered. Particular attention is directed at the problem of the diffusion "removal" or "injection" of certain elements which have a substantial influence

Card 1/3

ACCESSION NR: AT4034323

on the strength properties of the metals (e.g., lead penetration). It is pointed out that this process may change the creep characteristics and the long-term strength of the metal. Of less significance is the surface effect of the molten metal, this being, essentially, one of the forms of corrosion. In steels which are rather stable with respect to surface influences (fusion, adsorption), the long-term strength is frequently influenced primarily by the diffusion of certain elements. The transfer of these elements is related to their concentration gradient and, generally speaking, is a function of the stress state level. In the case of a liquid-metal environment, there is normally no perceptible dependence of the diffusion process on the stress state. This fact makes it possible to consider the phenomenon of diffusion separately from the stress field. The failure process, on the other hand, depends essentially on the stress state and on the element concentration level, which varies as a result of diffusion in the course of time. The author analyzes the problem of the failure of a circular rod (initial diameter  $2a$ ), extended by a force  $P$ . A solution is derived which permits the determination of the scale factor in the failure of rods of different diameters. Under identical physical conditions, this scale factor is shown to depend on the effect of the rod dimensions on the diffusion process. The fusion of the surface layer of the metal is stated by the author to be one of the forms of corrosion and to

Card 2/3

ACCESSION NR: AT4034323

depend, in general, on the stress state level. However, in the case of a liquid-metal medium, there is no difference between the potentials of the metal and the medium, as a result of which there are also no electrochemical processes. Under these conditions the effect of the stress state on the fusion process is negligible. The process of brittle failure of a rod gradually fused in a liquid-metal medium is analyzed. Surface cracks are shown to have a considerable effect, on occasion, on the strength of a body. In order to describe this process, the author suggests a phenomenological scheme which leads to the derivation of a specific formula for the process of brittle failure. Thus, while in this article, for the sake of simplicity problems relating to the elongation of a rod have been considered, the author notes that a similar analysis might be made for different problems; for example, pipe strength, rod bending and twisting, etc. "The author expresses his thanks to A. V. Stanyukovich for his comments." Orig. art. has: 20 formulas and 1 diagram.

ASSOCIATION: *Matematiko-mekhanicheskiy fakul'tet, Leningradskiy Universitet*  
(Department of Mathematical Mechanics, Leningrad University.)

SUBMITTED: 00

ENCL: 00

SUB CODE: MM

NO REF SOV: 005

OTHER: 000

Card 3/3

ACCESSION NR: AP4043890

S/0179/64/000/004/0063/0067

AUTHOR: Kachanov, L. M. (Leningrad)

TITLE: Creep of anisotropic solids

SOURCE: AN SSSR. Izvestiya. Mekhanika i mashinostroyeniye, no. 4, 1964, 63-67

TOPIC TAGS: metal creep, creep, plastic creep, anisotropic solid creep, anisotropic metal creep

ABSTRACT: Creep of anisotropic metals is considered, since the results are useful for solving problems concerning the creep of plastics. Anisotropic properties in relation to creep may be caused, for example, by rolling (or other plastic treatment methods), leading to changes of texture. While variations in texture have little effect on the coefficient of elasticity, the material remaining isotropic, they do affect strength and plastic properties. The established equation for residual creep of an isotropic solid is:

$$\dot{\epsilon}_{ij} = \frac{\partial \Lambda}{\partial \sigma_{ij}} \quad (i, j = x, y, z) \quad (1)$$

When T is the intensity of tangential stress and  $\Lambda$  is the additional dissipation:

$$6T^2 = (\sigma_x - \sigma_y)^2 + (\sigma_y - \sigma_z)^2 + (\sigma_z - \sigma_x)^2 + 6(\tau_{xy}^2 + \tau_{yz}^2 + \tau_{zx}^2) = 6T_0^2 \quad (2)$$

Card 1/4



ACCESSION NR: AP4043890

For an orthotropic solid, equation (2) is transformed into:

$$\sigma_T = A_1(\sigma_x - \sigma_y)^2 + A_2(\sigma_y - \sigma_z)^2 + A_3(\sigma_x - \sigma_z)^2 + 6(A_{12}\tau_{xy} + A_{23}\tau_{yz} + A_{31}\tau_{xz}) \quad (3)$$

When the main axes of stress coincide with the main axes of anisotropy, only the first three members remain in equation (3). The deformation rate is:

$$\dot{\epsilon}_x = \frac{1}{2}\Lambda^2 [A_1(\sigma_x - \sigma_y) + A_2(\sigma_x - \sigma_z)], \quad \dot{\gamma}_{xy} = 2\Lambda^2 A_{12}\tau_{xy} \quad (4)$$

On the basis of the evolved equations the author, using his previously published equation, finds that

$$N = \int THdV \quad (5)$$

$$N = (m + 1)\Lambda^2 = (l + 1)L^2 \quad (6)$$

Card 2/4.

ACCESSION NR: AP4043890

where  $N$  is the power of the given external forces. Irregular creep equations are then derived:

$$\left(\frac{\partial \sigma}{\partial t} + v\right) \frac{\partial \sigma}{\partial r} = \eta \dot{\epsilon} \quad \int (\Lambda + \frac{\partial \Pi}{\partial t}) dV = \min \quad (7)$$

The first of these shows the creep and the second shows the minimum additional dissipation. Sometimes the anisotropy is weak even with significant variation in the deformation rate. Equations for this case are also solved. An example is then given for a pipe under the influence of internal pressure. The radial velocity is found on the basis of noncompressibility:

$$b = \frac{c}{\lambda} + \frac{1}{2} \epsilon r \quad (8)$$

By transformations:

$$\sigma_r = \int \frac{R}{r} dr - p, \quad \sigma_\theta = \sigma_r + R \quad (9)$$

$$\int \frac{R}{r} dr - p = 0 \quad \text{at } r = b \quad (10)$$

Card 3/4

ACCESSION NR: AP4043890

where P is the axial force on the pipe. These equations are solved either numerically or by using curves. If the axial force caused by pressure at the pipe bottom  $P = p r a^2$ , then, when  $A_2 = A_3$ , the rate of axial elongation  $c$  equals zero. Orig. art. has: 29 equations.

ASSOCIATION; none

SUBMITTED: 06Mar64

ENCL: 00

SUB CODE: MM, MT

NO REF SOV: 002

OTHER: 003

Card 4/4

1 52543-25 ... /EPR/T/ENT(10) ...

ACCESSION NR: AP5010185

UR/0373/65/000/001/0050/0055

AUTHOR: Kachanov, L. M. (Leningrad)

26  
E

TITLE: Creep at oscillating temperature and load

SOURCE: AN SSSR. Izvestiya. Mekhanika, no. 1, 1965, 50-55

TOPIC TAGS: temperature field, creep mechanism, variational calculus, method, ...

ABSTRACT: The creep mechanism in a stable metallic body under the action of a slowly varying load and in a variable temperature field was investigated analytically. The temperature field is denoted by  $\theta = \theta_0 + \lambda \theta_1$  where  $\theta_0$  is the stationary field and  $\theta_1 = \theta_1(x, y, z, t)$ . The creep deformation rate is given by

$$\dot{\epsilon}_{ij} = \frac{1}{3} B(\theta) f_1(T) s_{ij} \quad (s_{ij} = \sigma_{ij} - c\delta_{ij}, T = (\frac{1}{2} s_{ij} s_{ij})^{1/2})$$

where  $B(\theta) = B_0 e^{\theta}$ . The secondary dissipation and the elastic potential are defined by

$$B(\theta) \int f_1(s) s ds \equiv B(\theta) \Lambda_1(T)$$

Card 1/3

ACCESSION NR: AP5010185

and

$$\Pi = \frac{3}{2} k\sigma^2 + \frac{1}{2G} T^2 \quad \left( k = \frac{1-2\nu}{E} \right)$$

respectively. These are incorporated into the creep equation which in its variational form is given by

$$\delta \int_V \left( A + \frac{\partial \Pi}{\partial t} + 3\alpha_0 \lambda \frac{\partial \theta}{\partial t} \right) dV = 0$$

The general solution is expressed by the sum of a stationary stress  $\sigma_{ij}^0$  and the auxiliary stress tensor  $\sigma'_{ij}$ . The expression under the integral sign above is expanded in powers of  $\lambda$  up to three terms (i.e., up to  $\lambda^2$ ). In this manner the auxiliary stress satisfies the variational equation

$$\delta \int_V \left[ \frac{dA_0}{dT_0^2} T'^2 + \frac{1}{2} \frac{d^2 A_0}{d(T_0^2)^2} N^2 + c_0 \frac{dA_0}{dT_0^2} N + \frac{\partial \Pi'}{\partial t} + 3\alpha_0 \lambda \frac{\partial \theta'}{\partial t} \right] dV = 0$$

The case of a fast and a slowly varying temperature field is discussed, and it is shown that for the latter case  $\sigma'_{ij}$  contains only the first two terms of the  $\lambda$  expansion. An approximate solution is proposed for the variational equation using the Ritz method. The problem is then extended to the case of a variable load  $F_n$  defined by

$$F_n = F_{n0} + \lambda F_{n1} \sin \omega t$$

Card 2/3

L 52543-00

ACCESSION NR: AP5010185

The corresponding variational equation takes the form

$$\delta \int \left[ \frac{dA_1}{d\tau} T^2 + \frac{1}{2} \frac{dA_2}{d(\tau^2)} N^2 - \frac{dH}{dt} \right] dt = 0$$

The cases of high and low frequency loads are discussed briefly and solutions proposed. Orig. art. has: 27 equations.

ASSOCIATION: none

SUBMITTED: 23Jul64

ENCL: 00

SUB CODE: ME

REF ID: A602

OTHER: 003

Card 3/3

BAKSHI, O.A.; KACHANOV, L.M.

Stressed state of a plastic interlayer in axisymmetric deformation.  
Izv. AN SSSR. Mekh. no.2:134-137 Mr-Ap '65.

(MIRA 13:6)

L 42310-60 EWI(d)/EWI(m)/EWP(w)/EWP(v)/EWP(k) IJF(c) NW/EM

ACC NR: AT6014514

SOURCE CODE: UR/2753/65/000/004/0065/0071

AUTHOR: Kachanov, L. M.

ORG: none

TITLE: Creep of momentless shells of revolution under large deformations

SOURCE: Leningrad. Universitet. Matematiko-mekhanicheskiy fakul'tet. Issledovaniya po uprugosti i plastichnosti, no. 4, 1965, 65-71

TOPIC TAGS: shell, shell of revolution, creep, creep rate, creep deformation, shell deformation, approximation method

ABSTRACT: The problem of large <sup>ll</sup> creep deformations of a <sup>pl</sup> thin momentless shell of revolution is studied under the effects of an internal pressure  $p$  and an axial force  $P$ . The values of  $p$  and  $P$  can, in general, vary with time. The stated problem is of interest for two reasons: 1) significant deformations sometimes occur during creep, and 2) analysis of large deformations is needed for determining the time of "viscous" failure. The problem is stated as follows: At an initial moment in time  $t = 0$  the median surface of the shell is given by the equation

$$r = r_0(z) \quad (0 < z < z_0)$$

where  $r, z$  are cylindrical coordinates, and the initial thickness of the shell is given by  $h_0 = h_0(z)$ . At a time  $t$  the median surface is characterized by

$$r = r(z, t),$$

Card 1/3

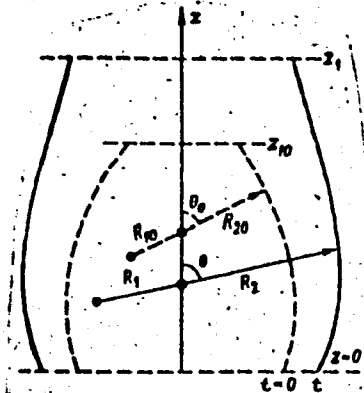


L 42310-66

ACC NR: AT6014514

where  $z$  is bounded by  $0 < z < z_1$ . Figure 1 shows the coordinate relationships for the deformed and undeformed shell.

Fig. 1.



Denoting the meridional and annular stresses as  $\sigma_1$  and  $\sigma_2$ , the principal radii of curvature as  $R_1$  and  $R_2$ , and the angle between the normal to the surface and the axis  $z$  as  $\theta$ , the author states the equilibrium and differential geometry conditions

$$\frac{\sigma_1}{R_1} + \frac{\sigma_2}{R_2} = \frac{p}{h};$$

$$2\sigma_1 h \sin \theta = p;$$

Card 2/3

ACC NR: AT6014514

$$R_1 = -\frac{1}{r^2} (1+r^2)^{1/2}; R_2 = r(1+r^2)^{1/2};$$

$$\sin \theta = \frac{1}{\sqrt{1+r^2}}; \operatorname{ctg} \theta = r;$$

An integral expression for the time to reach a particular state of deformation is derived. A "stepped" system of numerical integration may be used to solve this equation, or, in several cases, an approximate solution is obtainable from a variational equation of shell creep. Orig. art. has: 27 equations and 1 figure.

SUB CODE: 20/ SUBM DATE: 23Mar64/ ORIG REF: 004/ OTH REF: 001

Card 3/3 *bdh*

KACHANOV, N.F., inzh., red.; SHOPENSKIY, L.A., inzh., red.; IFTINKA, G.A., red.izd-va; MOCHALINA, Z.S., tekhn. red.

[Construction specifications and regulations] Stroitel'nye normy i pravila. Moskva, Gosstroizdat. Pt.2. Sec.G. ch.8. [Hot-water supply; standards of design] Goriachee vodosnabzhenie; normy proektirovaniia (SNiP II-G. 8-62). 1963. 11 p. (MIRA 16:9)

1. Russia (1923- U.S.S.R.) Gosudarstvennyy komitet po delam stroitel'stva. 2. Gosudarstvennyy komitet Soveta Ministrov SSSR po delam stroitel'stva (for Kachanov). 3. Nauchno-issledovatel'skiy institut sanitarnoy tekhniki Akademii stroitel'stva i arkhitektury SSSR (for Shopenskiy). (Hot-water supply)

KACHANOV, N.F., inzh.; SHOPENSKIY, L.A., inzh.

New standards for the design of hot-water supply. Vod.1 san.  
tekh. no.4:32-34 Ap '63. (MIRA 16:4)  
(Hot-water supply—Standards)

KOP'YEV, Sergey Fedotovich, prof., doktor tekhn. nauk; KACHANOV,  
Nikolay Filippovich, inzh.; ZAMYSHLYAYEVA, I.M., red.

[Principles of heat supply and ventilation] Osnovy teploga-  
zosnabzhenia i ventiliatsii. Moskva, Stroiizdat, 1964.  
227 p. (MIRA 17:8)

KACHANOV, N.F., inzh.; SHOPENSKIY, L.A.

Calculating the circulation pipelines of hot water supply systems  
in the direct supply of water from a heating network. Vod. i san.  
tekh. no.2:14-16 F '64 (MIRA 18:2)

KACHANOV, N.S., 1965, 27 p.

[Instruction on the selection of all-purpose centrifugal fans with A2 and A02 electric motors for sanitary engineering systems] Instruktsiia po podboru tsentrobeznykh ventilatorov obshchego naznacheniia s elektrodvigateliami serii A2 i A02 dlia sanitarno-tekhnicheskikh sistem. Moskva, No.2. 1965. 27 p. (MIRA 18:8)

1. Russia (1923. U.S.S.R.) Gosudarstvennyy komitet po delam stroitel'stva. Glavnoye upravleniye po stroitel'nomu proyektirovaniyu predpriyatii, zdaniy i sooruzheniy.

KACHANOV, N. N.  
Category : USSR/Solid State Physics - Structural crystallography

E-3

Abs Jour : Ref Zhur - Fizika, No 1, 1957, No 1102

Author : Kachanov, N.N.

Title : Instrument for Mechanical Determination of the Setup Parameters of Backward-Photography X-Ray Cameras

Orig Pub : Zavod. laboratoriya, 1955, 21, No 2, 241-242

Abstract : No abstract

Card : 1/1



КАЧАНОВ, Н.Н.

137-58-4-8525

Translation from: Referativnyy zhurnal, Metallurgiya, 1958 Nr 4, p 315 (USSR)

AUTHORS: Kachanov, N.N., Mirkin, L.I.

TITLE: X-ray Tubes for High-speed and Super-speed Photography and X-ray Research Procedures (Rentgenovskiye trubki dlya skorostnoy i sverkhskorostnoy s"yemki rentgenogramm v praktike rentgenostrukturnykh issledovaniy)

PERIODICAL: Tekhnol. avtomobilestroyeniya, 1957, Nr 5, pp 72-76

ABSTRACT: A survey. A brief description is presented of the design of Soviet and foreign sharp-focus tubes with punctuate and linear focus. The modes of operation of various parts for tubes with controllable focus, with magnetic focusing, with rotating plates, etc., are presented. It is noted that the use of sharp-focusing tubes permits exposure to be reduced by 98-98.4%. On pulse operation from batteries, exposure is reduced to 0.03-0.05 sec; this makes it possible to investigate fast-moving processes (structural changes in heating, impact stresses, etc.). Bibliography: 15 references.

Card 1/1

1. X-ray tubes--Design 2. X-ray photography--Equipment V.Sh.  
3. X-ray analysis--USSR

18.9100

67421

SOV/123-59-12-47059

Translation from: Referativnyy zhurnal. Mashinostroyeniye, 1959, Nr 12, p 159  
(USSR)

AUTHORS: Kachanov, N.N., Mirkin, L.I.

TITLE: X-ray Tubes for the High-speed and Superhigh-speed Taking of  
Roentgenograms in the Practice of X-ray Investigation

PERIODICAL: Tekhnol. avtomobilestroyeniye, 1957, Nr 5, pp 72-76

ABSTRACT: The authors describe sharp-focusing X-ray tubes (XT) with spot focus which make it possible to shorten considerably the time of exposure in carrying out X-ray analyses of metals and their alloys. Two versions of Soviet-made XT are mentioned, possessing an adjustable focus and a grounded cathode, with which focuses of very small dimensions can be obtained with a high specific load per surface unit of the anode. A description is also given of 2 XT with a trans-anode space, one of the XT possessing a magnetic focusing. Moreover an XT is described, with a particularly long and sharp line focus for investigating crystalline structures of monocrystalline specimens; with the aid of this tube convergent

Card 1/2

KACHANOV, N.N.

137-58-4-8519

Translation from: Referativnyy zhurnal, Metallurgiya, 1958, Nr 4, p 313 (USSR)

AUTHORS: Kachanov, N.N., Mirkin, L.I.

TITLE: An Ionization Method of X-ray Structural Analysis of Metals  
(Ionizatsionnyy metod dlya rentgenostrukturnogo analiza metallov)

PERIODICAL: Tekhnol. avtomobilstroyeniya, 1957, Nr 5, pp 76-79

ABSTRACT: An ionization method of recording X-ray radiation is described and comparison thereof with the photographic method is made. This method requires ionizing X-ray apparatus with an automatic recording arrangement. Ionizing X-ray apparatus have come into use in the investigation and inspection of materials used in the construction of machinery, phase analysis of metals and alloys, and also in the determination of residual stresses in metals. In studying the structure of the material, the level of accuracy is higher than by the photographic method, with a very pronounced reduction in the total time required for an investigation. This makes it possible to study transformations in alloys directly during the process of heat treatment or in the process of elastic and plastic deformation. Z.F.  
1. Metals--X-ray analysis 2. Metals--Phase studies 3. X-rays  
--Ionizing effects

Card 1/1